

MODELLI MECCANICI

1) $\sum F_{\text{mech}} = 0$

2) Tipologia delle forze meccaniche

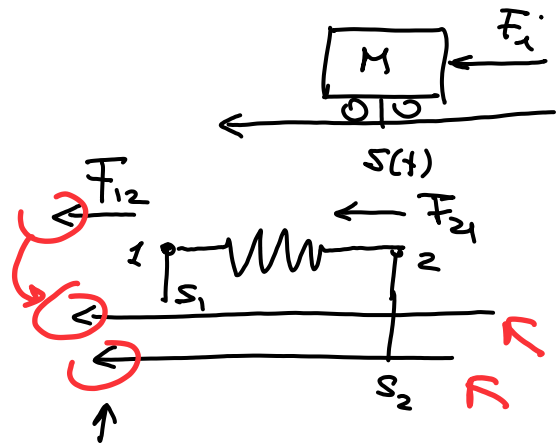
2A) Forza inerziale

$$F_i(t) = -M \ddot{s}(t)$$

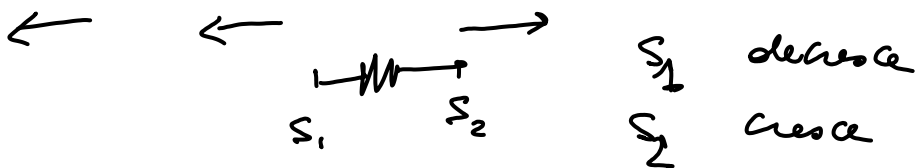
2B) Forza elastica

$$F_{12} = -k(s_1 - s_2)$$

$$F_{21} = -F_{12} = -k(s_2 - s_1)$$

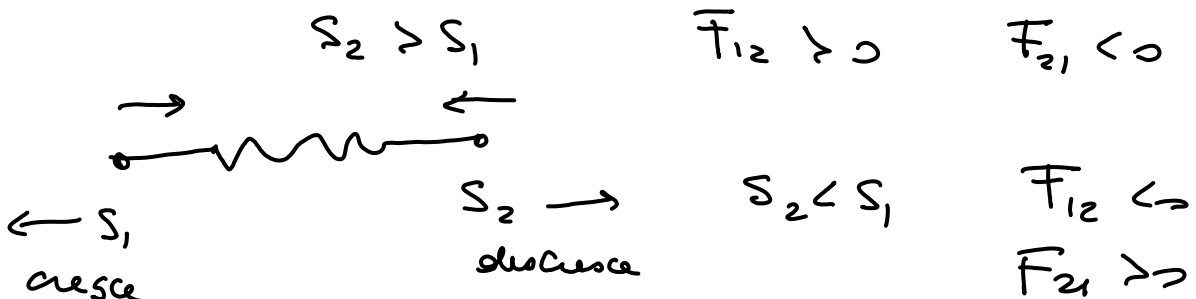


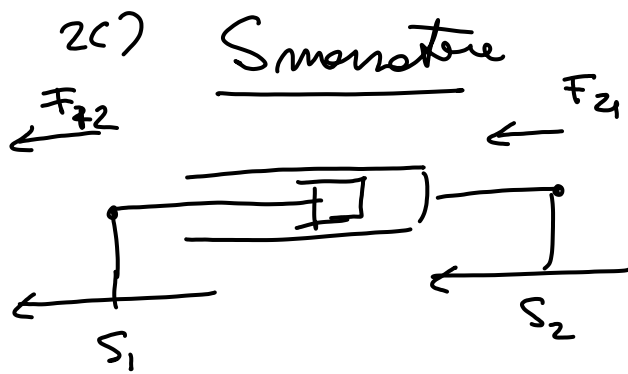
Esempio: molla centrata



$$s_2 < s_1$$

 molla usata



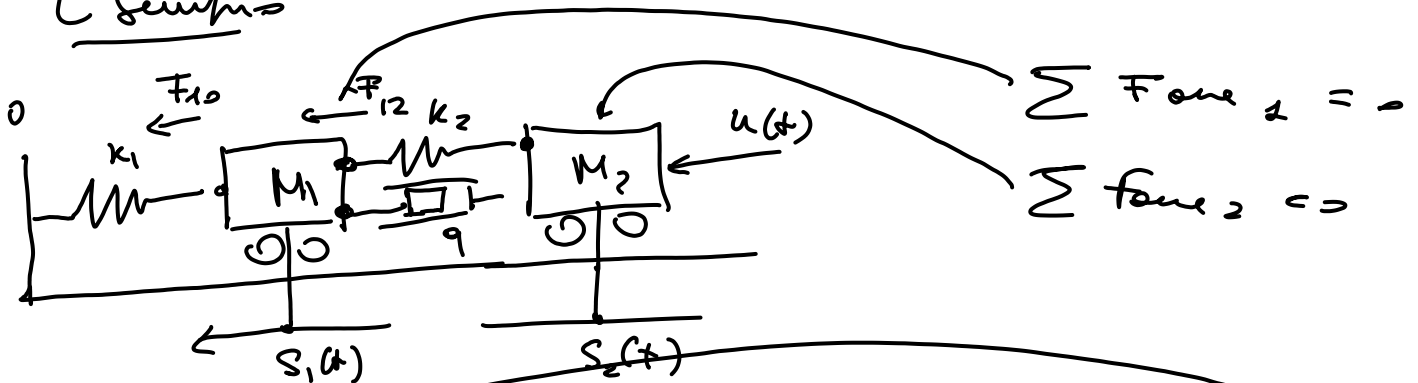


$$F_{12} = -q(\dot{s}_1 - \dot{s}_2)$$

$$F_{21} = -q(\dot{s}_2 - \dot{s}_1)$$

2d) Force external

Exemple



$$\sum \text{Force}_1 = 0$$

$$\sum \text{force}_2 = 0$$

$$\begin{aligned} -M_1 \ddot{s}_1 - k_1(s_1 - 0) - k_2(s_1 - s_2) - q(\dot{s}_1 - \dot{s}_2) &= 0 \\ -M_2 \ddot{s}_2 - k_2(s_2 - s_1) - q(\dot{s}_2 - \dot{s}_1) + u(t) &= 0 \end{aligned}$$

Modèle en état

$$x_1(t) = s_1(t) \quad x_2(t) = \dot{s}_1(t)$$

$$x_3(t) = s_2(t) \quad x_4(t) = \dot{s}_2(t)$$

$$\dot{x}_1 = x_2$$

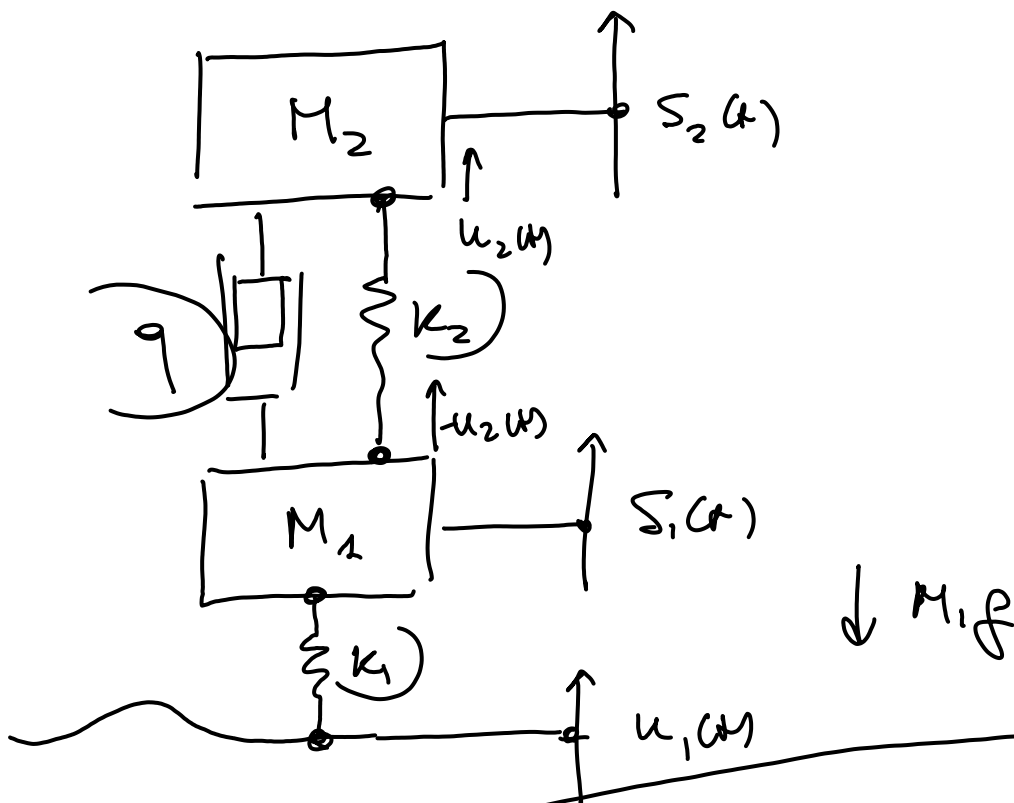
$$\dot{x}_2 = \ddot{s}_1 = \frac{1}{M_1} (-k_1 x_1 - k_2 x_1 + k_2 x_3 - q x_2 + q x_4)$$

$$\dot{x}_3 = x_4$$

$$\dot{x}_4 = \ddot{s}_2 = \frac{1}{M_2} (k_2 x_1 - k_2 x_3 + q x_2 - q x_4) + \frac{1}{M_2} u$$

$$4 \times 4 \quad A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{k_1+k_2}{M_1} & -\frac{q}{M_1} & \frac{k_2}{M_1} & \frac{q}{M_1} \\ 0 & 0 & 0 & 1 \\ \frac{k_2}{M_2} & \frac{q}{M_2} & -\frac{k_2}{M_2} & -\frac{q}{M_2} \end{bmatrix} \quad B = \begin{bmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{M_2} \end{bmatrix}$$

Esempio: modello di una sospensione



$$\begin{aligned} -M_1 \ddot{s}_1 - k_1(s_1 - u_1) - k_2(s_1 - s_2) - q(\dot{s}_1 - \dot{s}_2) \\ - u_2(t) - M_1 g = 0 \end{aligned}$$

$$-M_2 \ddot{s}_2 - k_2(s_2 - s_1) - q(\dot{s}_2 - \dot{s}_1) + u_2 - M_2 g = 0$$

Modello in forme ai stato

Punto fermo previsto $= 0$

$$x_1(t) = S_1(t)$$

$$\underline{\underline{x_2(t) = \dot{S}_1(t)}}$$

$$\dot{x}_3(t) = S_2 \leftarrow$$

$$x_4 = \dot{S}_2$$

$$\dot{x}_1 = x_2$$

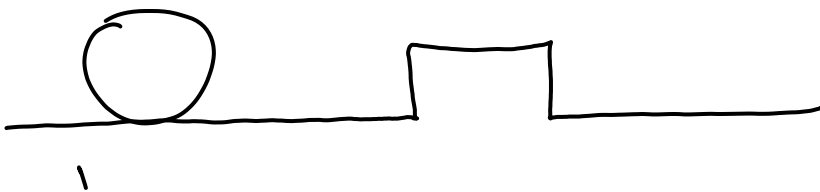
$$\dot{x}_3 = x_4$$

$$\underline{\dot{x}_2 = \ddot{S}_1} = \frac{1}{M_1} \left(-k_1 x_1 - k_2 x_1 - q x_2 + k_2 x_3 + q x_4 \right) - \frac{1}{M_1} u_2 + \frac{k_1}{M_1} u_1$$

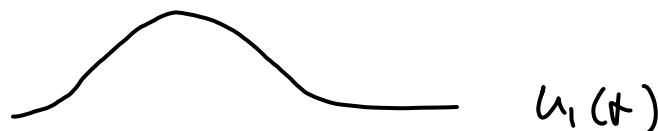
$$\underline{\dot{x}_4 = \ddot{S}_2} = \frac{1}{M_2} \left(k_2 x_1 + q x_2 - k_2 x_3 - q x_4 \right) + \frac{1}{M_2} u_2$$

$$A = \begin{bmatrix} & & & \\ & & & \\ & & & \\ & & & \end{bmatrix}$$

$$B = \begin{bmatrix} 0 & 0 \\ k_1/M_1 & -1/M_1 \\ 0 & 0 \\ 0 & 1/M_2 \end{bmatrix} \begin{matrix} \leftarrow \\ \leftarrow \end{matrix}$$



S_2



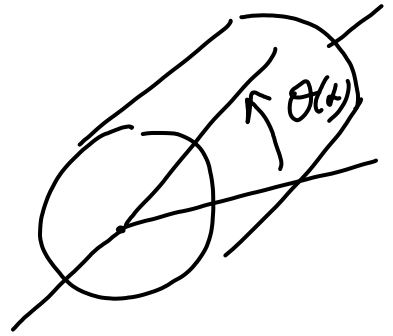
Sistemi meccanici in moto rotatorio

1) $\sum \text{coppie} = 0$

2) 2A) Coppia inerziale

$$C_i(t) = -J \ddot{\theta}(t)$$

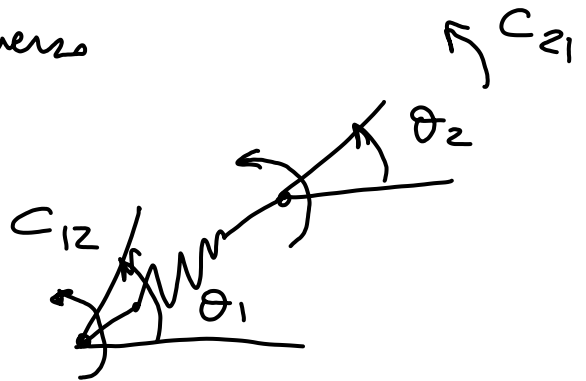
$\uparrow$
momento di inerzia



2B) Coppia elastica

$$C_{12} = -k(\theta_1 - \theta_2)$$

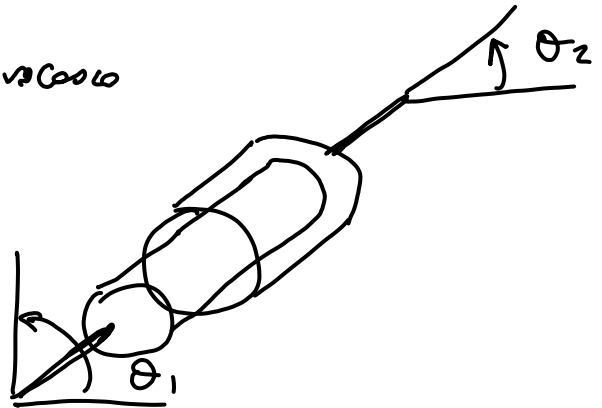
$$C_{21} = -k(\theta_2 - \theta_1)$$



3C) Coppia di attrito viscoso

$$C_{12} = -q(\dot{\theta}_1 - \dot{\theta}_2)$$

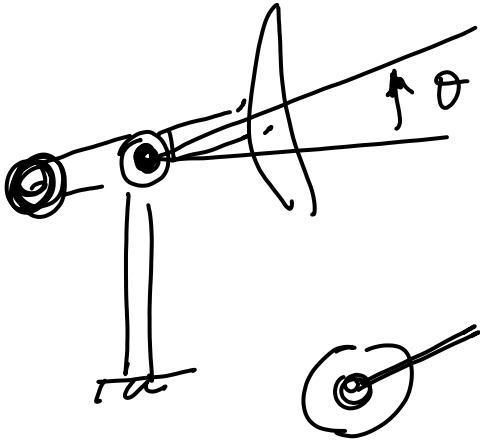
$$C_{21} = -q(\dot{\theta}_2 - \dot{\theta}_1)$$



3D) Coppie esterne

Esercizio

Puntamento di un radar



$$-J\ddot{\theta} - q(\dot{\theta} - 0) + C_M + C_V = 0$$

$u_1 \quad u_2$

C_M coppia generata
dal motore elettrico

C_V coppia generata dal
vento

$$x_1 = \theta$$

$$x_2 = \dot{\theta}$$

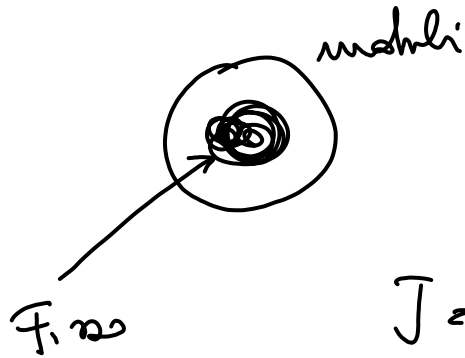
$$\dot{x}_1 = x_2 \quad \leftarrow$$

$$\dot{x}_2 = \ddot{\theta} = \frac{1}{J} (-q x_2 + u_1 + u_2)$$

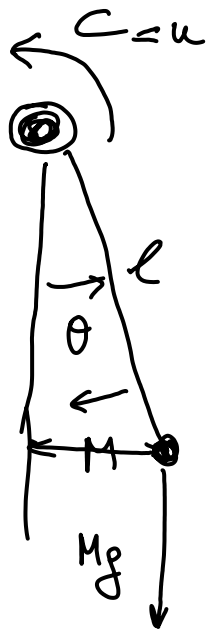
$$A = \begin{bmatrix} 0 & 1 \\ 0 & -q/J \end{bmatrix}$$

$$B = \begin{bmatrix} 0 & 0 \\ 1/J & 1/J \end{bmatrix}$$

Pendulo



$$J = l^2 M$$



$$-J \ddot{\theta} - q \dot{\theta} = M g l \sin \theta + u = 0$$

$$x_1 = \theta$$

$$x_2 = \dot{\theta}$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \frac{1}{J} (-M g l \sin(x_1) - q x_2 + u)$$

Se (x_1) piccolo

$$\sin(x_1) \approx x_1$$

$$\dot{x}_2 \approx -\frac{M g l}{J} x_1 - \frac{q}{J} x_2 + \frac{1}{J} u$$

Equilibrio = soluzioni costanti

$$u(t) = \bar{u} = 0$$

$$\theta(t) = \bar{\theta} \text{ costante}$$

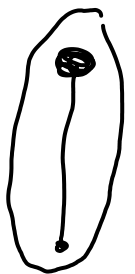
$$-J \bar{\theta} - q \bar{\theta} - M g l \sin \bar{\theta} + \bar{u} = 0$$

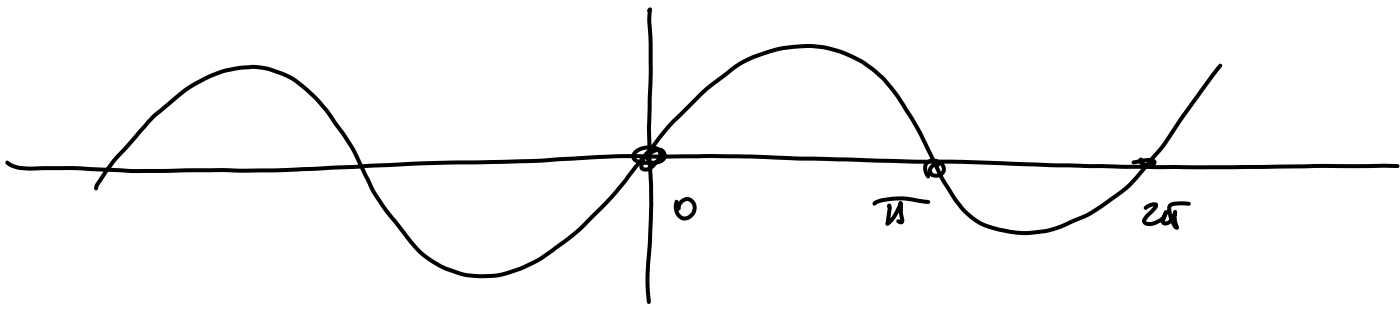
$$A = \begin{bmatrix} 0 & 1 \\ -\frac{M g l}{J} & -\frac{q}{J} \end{bmatrix}$$

$$\sin \bar{\theta} = 0$$

$$\bar{\theta} = 0$$

$$\bar{\theta} = \pi$$





Sei $\bar{\theta} = \pi$ die konstante neue
variable $\tilde{\theta}(t) \stackrel{\text{def}}{=} \theta(t) - \bar{\theta} = \theta(t) - \pi$

$$-J\ddot{\theta} - b\dot{\theta} - Mgl \sin \theta + u = 0$$

$$\theta(t) = \bar{\theta} + \tilde{\theta}(t)$$

$$\frac{d^2}{dt^2} (\bar{\theta} + \tilde{\theta}(t))$$

$$= \frac{d^2}{dt^2} \bar{\theta} + \frac{d^2}{dt^2} \tilde{\theta} = \ddot{\tilde{\theta}}$$

Operable $\sin \theta \approx \theta$

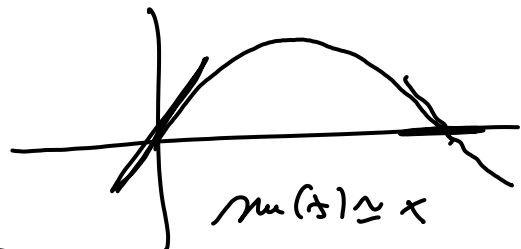
$$-J\ddot{\tilde{\theta}} - b\dot{\tilde{\theta}} - Mgl \underbrace{\sin(\pi + \tilde{\theta})}_{\approx -\tilde{\theta}} + u = 0$$

$$\tilde{\theta} \approx 0$$

$$f(x) \approx \underline{f(\bar{x})} + \underline{f'(\bar{x})} \hat{x}$$

$$\sin(\pi + \tilde{\theta})$$

$$\approx \cancel{\sin(\pi)} + \cos(\pi) \tilde{\theta} = -\tilde{\theta}$$



$$-J\ddot{\tilde{\theta}} - b\dot{\tilde{\theta}} + Mgl \tilde{\theta} + u = 0$$

$$x_1 = \tilde{\theta}$$

$$x_2 = \dot{\tilde{\theta}}$$

$$\dot{x}_1 = x_2$$

$$A = \left[\begin{array}{c|c} 0 & 1 \\ \hline M_{ge}/J & -b/J \end{array} \right] \quad B = \begin{bmatrix} 0 \\ 1/J \end{bmatrix}$$

$$\dot{x}_2 = \ddot{\theta} = \frac{1}{J} (\underline{M_{ge}} x_1 - \underline{b} x_2 + u)$$
